

What brakes can tell you

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Outline

- 1 Motivation
- 2 The Experiment
- 3 Model
- 4 Two Pin Problem

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Motivation for the Model

- Movement from old drum brake system to new disc brake system with improved stopping distances.
- Can still be seen to this day when comparing the American and European HGV industries where improving regulations require the disc brake system.
- However, disc brakes more prone to contamination and squeal.
- Thus, we would like to model the brake system and deformation at a small level.

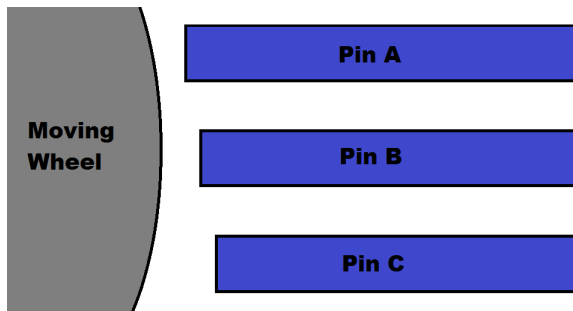
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J. R. Barber's Experiment

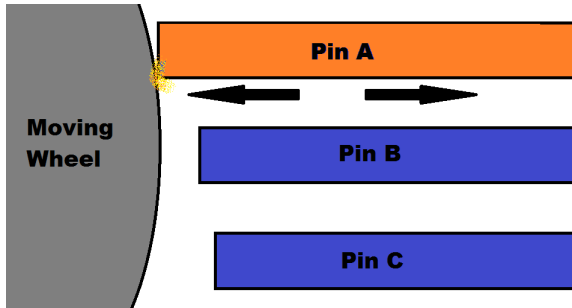
- Initial Experiment with full brake.
 - Unstable resonant vibration due to only small areas of contact.
 - In turn due to microscopic rough nature of surface accentuated by thermal deformation.
- To model this Barber devised a simple experiment:
 - 3 pins attached to a loading arm to act as a brake.
 - Pins could then be pushed against a fast moving wheel to simulate braking.
 - Each pin implanted with thermocouple to measure the temperature.

Barber's Observations and Explanation



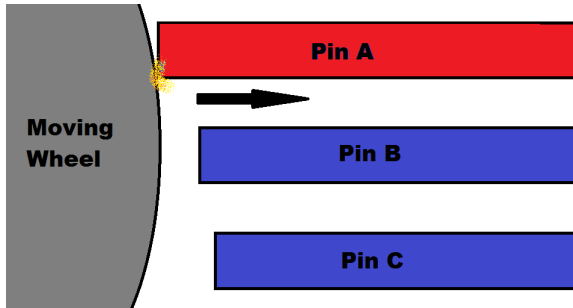
- Three cold pins at slightly different heights to mimic the microscopic jaggedness of the brake pad.

Stage 1



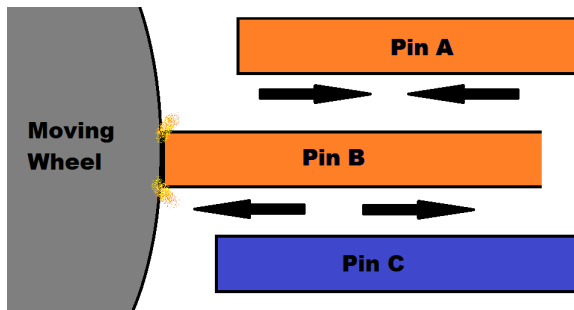
- The wheel makes contact with pin A.
- Temperature observed from the thermocouple associated with pin A rises.
- Pin A expands.
 - Temperature still relatively low - thermal expansion outweighs the wear.
 - Initial irregularity of surface of brake exaggerated.

Stage 2



- Temperature of pin reaches equilibrium - heat generated due to friction balanced by heat lost to surroundings. At this point the thermal expansion stops.
- Rate of wear on pin increases with temperature.
 - a time will be reached at which the increase in height of pin A due to thermal expansion is superseded by the decrease in height of the pin due to wear.

Stage 3



- Increase in wear means a change in contact point to pin B is inevitable.
- The pin has a lower wear rate and a higher rate of expansion - cycle is started again.
- Pin A cools and shrinks making it the smallest of the three pins.

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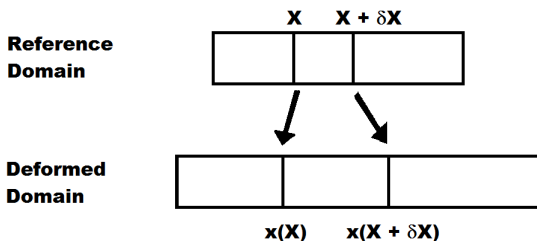
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Model Derivation: Setting

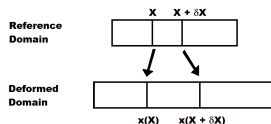
- Pin is modelled as 1D elastic material and is subject to physical laws of elasticity.
- Define displacement $u(\mathbf{X})$ of a material point \mathbf{X} to be given by

$$u(\mathbf{X}) = x(\mathbf{X}) - \mathbf{X}$$

where $x = x(\mathbf{X})$ is spatial location of material point \mathbf{X} once the pin has been deformed.



Model Derivation: Hooke's law



- Define $l_0 := (\mathbf{X} + \delta\mathbf{X}) - \mathbf{X} = \delta\mathbf{X}$ and $l := x(\mathbf{X} + \delta\mathbf{X}) - x(\mathbf{X})$.
- In **isothermal** case,

$$\tau = k \frac{l - l_0}{l_0} \quad (\text{"stress"} \propto \text{"strain"}).$$

Model Derivation: Hooke's law

- Localising Hooke's law, we have

$$\begin{aligned}\tau(\mathbf{X}) &= k \lim_{\delta\mathbf{X} \rightarrow 0} \frac{x(\mathbf{X} + \delta\mathbf{X}) - x(\mathbf{X}) - (\mathbf{X} + \delta\mathbf{X} - \mathbf{X})}{\delta\mathbf{X}} = k \frac{\partial u}{\partial \mathbf{X}} \\ &= k \left(\frac{\partial x}{\partial \mathbf{X}} - 1 \right).\end{aligned}$$

- Under small strain assumption, we have

$$\left| \frac{\partial u}{\partial \mathbf{X}} \right| \ll 1 \Rightarrow \frac{\partial x}{\partial \mathbf{X}} \approx 1 \Rightarrow \frac{\partial u}{\partial \mathbf{X}} = \frac{\partial u}{\partial \mathbf{x}}.$$

- Can derive all equations in terms of more familiar spatial (*Eulerian*) coordinates x rather than material (*Lagrangian*) coordinates \mathbf{X} .

Model Derivation: Thermal Expansion

- Elastic materials subject to **thermal conditions without stress** satisfy

$$\frac{\partial u}{\partial x} = \alpha(T - T_0) \text{ ("strain" } \propto \text{ "change in temperature")}.$$

Model Derivation: Linear Model

- Hooke's law (Isothermal model):

$$\tau = k \frac{\partial u}{\partial x} \text{ ("stress" } \propto \text{ "strain")}.$$

- Thermal expansion law (no stress model):

$$\frac{\partial u}{\partial x} = \alpha(T - T_0) \text{ ("strain" } \propto \text{ "change in temperature")}.$$

- Have both thermal forcing and stress in pin so model strain as being a *linear combination of both stress and the change in temperature*:

$$\frac{\partial u}{\partial x} = \frac{1}{k} \tau + \alpha(T - T_0).$$



Model Derivation: Thermoelasticity equation

- Assume domain of pin is $\Omega = [0, 1]$, friction with wheel occurring at $x = 0$.
- At $x = 1$, $\tau = -F_0$ where F_0 is the force resulting from applying the brakes.
- Conservation of momentum $\Rightarrow \frac{\partial \tau}{\partial x} = 0 \Rightarrow \tau = -F_0$ for all $x \in [0, 1]$.
- Re-scaled so that $T_0 = 0$, we obtain *thermoelasticity equation*

$$\frac{\partial u}{\partial x} = \alpha T - \frac{1}{k} F_0.$$

Model Derivation: Heat conduction model

- Also pose a heat conduction model on pin which models diffusion of heat into pin due to contact with wheel at $x = 0$.
- Pointwise conservation of energy yields

$$\frac{\partial T}{\partial t} + a \frac{\partial^2 u}{\partial x \partial t} = \kappa \frac{\partial^2 T}{\partial x^2}$$

where $a > 0$ is small. The term $a \frac{\partial^2 u}{\partial x \partial t}$ takes into account expansion of elastic material as heat gets diffused through the material.

- Very similar model considered in Copetti & Elliott (1993).

Model Derivation: Heat Conduction IC/BC

- Need to prescribe physically appropriate initial/boundary conditions for thermoelasticity equation and heat conduction model satisfied by u and T at $x = 0, 1$.
- $\kappa \frac{\partial T}{\partial x} = 0$ at $x = 1$ (no heat flow at opposite end).
- Frictional heating \propto stress \times velocity:

$$\kappa \frac{\partial T}{\partial x} = \gamma \tau = -\gamma F_0 \quad \text{at } x = 0.$$

- The initial condition for the temperature is chosen to be

$$T(x, 0) = 0 \quad \text{for all } x \in [0, 1].$$

Model Derivation: Thermoelasticity BC

- Only need one boundary condition for well-posedness of thermoelastic equation.
- Wear of break pad due to the friction with wheel \propto stress \times velocity:

$$u(0, t) = -\beta F_0 t \quad \text{at } x = 0.$$

Decoupled Problem

- Recall

$$\frac{\partial u}{\partial x} = \alpha T - \frac{1}{k} F_0. \quad (1)$$

- Differentiating with respect to t , we get

$$\frac{\partial^2 u}{\partial x \partial t} = \alpha \frac{\partial T}{\partial t}.$$

- Plugging into heat conduction model, we obtain

$$\frac{\partial T}{\partial t} = \frac{\kappa}{(1 + a\alpha)} \frac{\partial^2 T}{\partial x^2}. \quad (2)$$

- Solution algorithm for the one pin problem can be summarised as follows:
 - Find temperature T satisfying (2) along with B.C./I.C.
 - Construct right-hand side of (1).
 - Find displacement u satisfying (1) along with B.C.

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Two Pin Problem - Setup

- We now apply the model to two pins of differing lengths.
- Assume pin 1 is longer than pin 2 and only the former is in contact with the plate at time $t = 0$.
- Wearing will cause pin 1 to shrink in length.

At some time $t = t^*$ pin 1 will have shrunk to point where lengths of two pins equal.

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Time for a little simulation...

Acknowledgements

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