

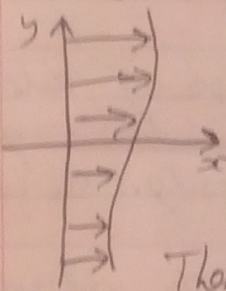
12pm 6th May 2014
B3.03

Fluids Revision 1

- Class 1: laminar pipe and shear flows.
 Class 2: Monday 12th 10-11 B3.03 - overview (question EMAIL)
 Class 3: Friday 23rd 11-12 B3.02 - Kerr
 Section 6 questions in revision questions.
 Rain fall on roof questions not examinable.

Shear Flow

We consider a velocity field $\underline{u} = (u(y,t), 0, 0)$
 with only one velocity in streamwise direction x ,
 which depends on y .



Incompressibility satisfied $\frac{d\rho}{dt} = 0 = \nabla \cdot \underline{u}$

Advection is zero $(\underline{u} \cdot \nabla) \underline{u} = u \frac{d}{dx} u = 0$

Pressure gradient is $\nabla P = (\frac{dP}{dx}, 0, 0)$

Viscosity dependency: $\nu \Delta u = \frac{d^2}{dy^2} u$

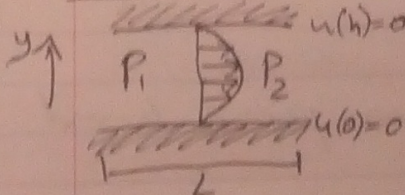
Then equation is: $\frac{d^2 u}{dy^2} = -\frac{1}{\rho} \frac{dP}{dx}$

Assume $\frac{dP}{dx} = P_x$ constant and assume steady state
 then solve $\frac{d^2}{dy^2} u = \frac{P_x}{\rho \nu}$

Integrate w.r.t y twice: $u(y) = \frac{P_x}{2\rho\nu} y^2 + Ay + B$

Poiseuille Flow

Flow through pipe



Pressure gradient is $\frac{dP}{dx} = \frac{P_2 - P_1}{L} = P_x$

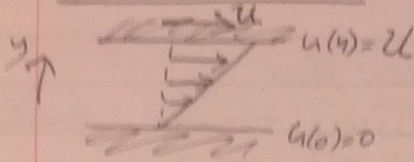
$u(0) = 0 \Rightarrow B = 0$

$u(h) = 0 \Rightarrow A = -\frac{P_x h}{2\rho\nu}$

$\Rightarrow u(y) = \frac{P_x}{2\rho\nu} y(y-h)$

Couette Flow

Flow between moving plates



Not pressure driven

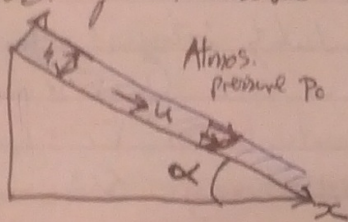
$$u(0) = 0 \Rightarrow B = 0$$

$$u(h) = U \Rightarrow A = \frac{U}{h}$$

$$\Rightarrow u(y) = \frac{U}{h} y$$

For large Reynolds numbers, Poiseuille / Couette flow turbulent.
Flow over an inclined plane

Layer of incompressible fluid bounded below by a plane fixed at an angle α to the horizontal and from above by a free surface.



Consider steady state due to balance between gravity and viscosity. $u = (u(y), 0, 0)$

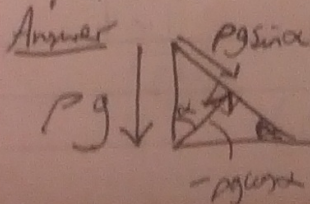
- a) Formulate no-slip boundary condition for velocity at the bottom plane, and pressure BC. at free surface.

Answer $u = v = \frac{du}{dy} = 0$ at $y = 0$
 \uparrow
 no flow through surface

- b) Why is $\frac{du}{dy} = 0$ at $y = h$

Answer Shear stress $\tau = \mu \frac{du}{dy}$ must be zero at the free surface (no fluid above), therefore $\frac{du}{dy} = 0$.

- (c) What are the gravitational forces perpendicular and parallel to the surface.



d) Give the steady state Navier-Stokes equations for this system.

Answer If $\mathbf{u} = (u(y), v(y), 0)$ then by incompressibility $\frac{\partial_x u + \partial_y v = 0 \Rightarrow \partial_y v = 0$ and $v(0) = 0 \Rightarrow v = 0$ everywhere.

So NS are: $0 = \underset{x \text{ direction}}{-\frac{1}{\rho} \frac{\partial P}{\partial x} + \nu \frac{\partial^2 u}{\partial y^2} + g \sin \alpha}$ (1)

$y \text{ direction} \rightarrow 0 = \frac{-1}{\rho} \frac{\partial P}{\partial y} - g \cos \alpha$ (2)

e) Solve for pressure.

Answer Integrate (2): $p(x, y) = -\rho g y \cos \alpha + f(x)$

Also $P_{x=h} = P_0$ at $y=h$ b.c.

subtract $p(x, h) = P_0 = -\rho g h \cos \alpha + f_0$ ($f(x)$ constant)
 $\Rightarrow p - P_0 = \rho g (h - y) \cos \alpha$ (3)

f) Solve velocity

Answer From (3) we see $\frac{\partial P}{\partial x} = 0$ so (1) reduces to $\nu \frac{\partial^2 u}{\partial y^2} = -g \sin \alpha$. Integrate and use BCs $u = 0$ at $y = 0$ and $\frac{\partial u}{\partial y} = 0$ at $y = h$

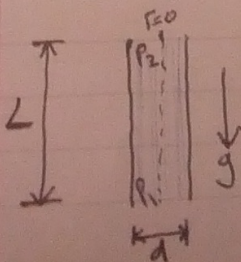
to get $u = \frac{g}{2\nu} y (2h - y) \sin \alpha$

g) Find volume of fluid Q passing through cross-section of the layer per unit length in the x -direction per unit time.

Answer $Q = \int_0^h u dy = \int_0^h \left[\frac{g}{2\nu} y h - \frac{g}{2\nu} y^2 \right] \sin \alpha dy$
 $= \left[\frac{g}{2\nu} h^2 y - \frac{g}{6\nu} y^3 \right] \sin \alpha \Big|_0^h = \frac{g}{3\nu} h^3 \sin \alpha$

Flow Through Pipes - Viscometer

Consider vertical pipe, assume cylindrically symmetric, steady, laminar Poiseuille flow.



Aim: show flow rate through pipes

$Q = \frac{\pi d^4}{128 \nu} \left(\frac{P_2 - P_1}{\rho L} + g \right)$

$$(\Delta = \frac{1}{r} \frac{\partial}{\partial r} r \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2})$$

N-S simplifications

(a) Navier-Stokes simplifies to (neglect time deriv, advection)

$$(4) \frac{d}{dr} \left(r \frac{dw}{dr} \right) = -\frac{r}{\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) \quad \text{using cylindrical coord.}$$

Find velocity profile.

(b) Flow is symmetric at $r=0$, BCs: $\frac{dw}{dr} = 0$ at $r=0$
also note that $w=0$ at $r=\frac{d}{2}$.

Integrate (4) to get

$$r \frac{dw}{dr} = -\frac{r^2}{2\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) + C_1$$

use $\frac{dw}{dr} = 0$ at $r=0$ to get $C_1 = 0$.

Integrate again $w = -\frac{r^2}{4\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) + C_2$

use $w=0$ at $r=\frac{d}{2}$ to get $C_2 = \frac{d^2}{16\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right)$

So that $w = \frac{1}{4\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) \left(\frac{d^2}{4} - r^2 \right)$

Find flow rate.

$$\begin{aligned} (c) \text{ Flow rate } Q &= 2\pi \int_0^{d/2} w r dr \\ &= 2\pi \int_0^{d/2} \frac{1}{4\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) \left(\frac{d^2 r}{4} - r^3 \right) dr \\ &= \left[\frac{2\pi}{4\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) \left(\frac{d^2 r^2}{8} - \frac{r^4}{4} \right) \right]_0^{d/2} \\ &= \frac{\pi}{2\nu} \left(\frac{p_2 - p_1}{\rho L} + g \right) \frac{d^4}{64} \end{aligned}$$